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NUMERICAL SIMULATIONS OF LOW MACH COMPRESSIBLE TWO-PHASE FLOWS: PRELIMINARY ASSESSMENT OF SOME BASIC SOLUTION TECHNIQUES*,**

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Abstract. The objective of this paper is to give a preliminary assessment of several existing approaches (compressible or incompressible, preconditioned or not preconditioned) to the numerical simulation of low Mach compressible two-phase flow problems in more than one space dimension. We consider a popular broken dam problem of Martin and Moyce [Philos. Trans. Roy. Soc. London A, 244(1952), pp. 312-324] as our first benchmark test where the approximate results obtained using each of numerical methods concerned here can be compared with the experiments for the basic solution validation. Other computations are also presented that show the accuracy as well as the performance that are attained using different numerical techniques.

Résumé. Notre objectif ici est de faire, dans le contexte de la simulation numerique d'ecoulements diphasiques a faible nombre de Mach, une evaluation preliminaire de plusieurs approches numeriques 2D basees sur des modeles incompressible ou incompressible (avec ou sans preconditionnement faible Mach). Les comparaisons sont faites sur deux problemes de references. Le premier cas test est celui de l'effondrement d'une colonne d'eau (Martin and Moyce [Philos. Trans. Roy. Soc. London A, 244(1952), pp. 312-324]). Le second cas test est celui d'une onde solitaire arrivant sur un rivage en forme de marche. Les methodes sont evaluees par rapport aux mesures experimentales et demontrent les performances et les precisions numeriques des differentes techniques numeriques utilisees.

1. Introduction

We are interested in solving unsteady weakly compressible two-phase flow problems where the flow speed is assumed to be much less than the sound speed of the fluid component of concerned and the wavelengths of the acoustic waves are assumed to be large. Representative applications of this kind of problems are such as the rising of gas bubbles in liquids, the falling of liquid drops in the air under gravitational force field, and bubbly flow in liquids, see [13, 36, 46] for more examples.

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It is without question that one possible approach to simulate the aforementioned low speed (single- or two-phase) flow problem is to consider it as a fully compressible flow and use a standard upwind finite volume method for numerical approximation (cf. [20]). When this is done with the use of an explicit method, it is known in the literature (cf. [11, 33–35, 44, 45, 48]) that we would have a severe time step restriction due to the CFL (Courant-Friedrichs-Lewy) condition for stability, yielding difficulties as the lack of robustness of the method and also the loss of accuracy of the computed solutions.

To overcome these difficulties, a common practice in many improved compressible flow solvers is to use splitting. That is, we separate the flow variables into various parts, and identify the "ill-conditioned" part of the flow variable (typically density or pressure) that may require some special "implicit" treatments. When this is done in a suitable manner, it is possible to bypass the stringent CFL condition dominated by the sound speed, and use a more robust flow speed-based CFL condition for the computations. Method of this type is first described by Harlow and Amsden [12], and later generalized by many other researchers (cf. [3,7,32,41,50] and references cited therein).

Besides the above approach, Colella and Pao [9] have proposed a projection-type method in that the compressible Euler equations were rewritten in terms of the combination of a Hodge decomposition of the velocity field and auxiliary pressures. With the new equations, the flow can be separated into the divergence-free part where the solution of the equations would vary on a time scale determined by the flow speed, and a part that the solution of the equations would depend on fast acoustic waves. Hence, the former part may be advanced with time step determined solely by the flow speed, and the latter part may be treated by an implicit method. In addition to that, following the low Mach number asymptotic analysis of Klainermann and Majda [17], Schneider et al. [33] have developed an asymptotic-based method in that elliptic constraints have been introduced to the construction of numerical flux functions for a class of finite volume compressible flow solvers that is a viable approach when the Mach number goes to zero. Lastly, in the work of Guillard and coworkers [11,28] a preconditioning technique has been used to the construction of flux functions that is an extension of the approach of given by Turkel [44,45] so as to have the desire asymptotic behavior of the method to the continuum model in the zero Mach number limit.

We note that many of the low speed flow solvers mentioned above are devised for solving single phase compressible flow problems only. The successful extension of the approaches among them to multiphase flow problems include the preconditioned method of Murrone and Guillard [28] and the constrained interpolation profile method of Yabe *et al.* [50]. It is our goal in this work to explore further some of these low speed approaches in the hope to come up with a robust and accurate numerical method for general weakly compressible multiphase flow with real materials.

As a first endeavor towards the method development, we consider a popular broken dam problem of Martin and Moyce [24] as our first benchmark test, and present results obtained using a sample numerical approaches based on incompressible or compressible solver for the basic solution validations, see the similar work done by Helluy et al. [13] for the numerical simulations of wave breaking which is an example of a low speed flow problem. Other computations will be done also that show the accuracy as well as the performance that are attained using different numerical techniques.

The format of this paper is as follows. In Section 2, we review briefly a fluid-mixture type algorithm compressible flow solver for homogeneous two-phase flows. In Section 3, we describe a preconditioned compressible two-phase solver with surface tension effect included in the problem formulation, and in Section 4, we give an incompressible flow solver based on the artificial compressibility approach. Numerical results are presented in Section 5.

2. A Compressible homogeneous two-phase flow solver

We begin our discussion by considering a simplified compressible two-phase flow problem in two space dimensions where there is a material interface separating regions of two different fluid components consisting of gases and liquids. In this problem, the flow regime of interest is assumed to be homogeneous with no jumps in the pressure and velocity across the interface, and the physical effects such as the viscosity, surface tension, and heat conduction are assumed to be small, and hence can be ignored in the problem formulation. With that, on the gas-phase part of the domain, the fluid is governed by the compressible Euler equations of the form

$$\frac{\partial}{\partial t} \begin{pmatrix} \rho \\ \rho u \\ \rho v \\ \rho E \end{pmatrix} + \frac{\partial}{\partial x} \begin{pmatrix} \rho u \\ \rho u^2 + p \\ \rho u v \\ \rho E u + p u \end{pmatrix} + \frac{\partial}{\partial y} \begin{pmatrix} \rho v \\ \rho u v \\ \rho v^2 + p \\ \rho E v + p v \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \\ -\rho g \\ -\rho g v \end{pmatrix}, \tag{1}$$

where ρ is the density, u and v are the velocities in the x- and y-direction respectively, p is the pressure, E is the total energy per unit mass, and g is the gravitational acceleration. We assume that the equation of state for the gas satisfies the ideal gas law

$$p = (\gamma - 1)\rho e,\tag{2}$$

where e is the specific internal energy, γ is the usual ratio of specific heats ($\gamma > 1$), and so $E = e + (u^2 + v^2)/2$. Note that the four components of Eqs. (1) express the conservation of mass, momenta in the x- and y-direction, and energy, respectively [10].

On the liquid-phase part of the domain, however, we assume the fluid to be barotropic, and use the Tait equation of state

$$p(\rho) = (p_0 + \mathcal{B}) \left(\frac{\rho}{\rho_0}\right)^{\gamma} - \mathcal{B} \tag{3}$$

as the constitutive law for the liquid. Here p_0 and ρ_0 are the reference pressure and density, respectively, and \mathcal{B} is a pressure-like constant, see [23, 29, 51] for a typical set of data of practical importance. It is clear that because the pressure is a function of the density only, the energy equation of the Euler Eqs. (1) plays no active role to the determination of this flow behavior, and hence can be neglected. For completeness, we write down the equations of motion for the liquid as follows,

$$\frac{\partial}{\partial t} \begin{pmatrix} \rho \\ \rho u \\ \rho v \end{pmatrix} + \frac{\partial}{\partial x} \begin{pmatrix} \rho u \\ \rho u^2 + p(\rho) \\ \rho u v \end{pmatrix} + \frac{\partial}{\partial y} \begin{pmatrix} \rho v \\ \rho u v \\ \rho v^2 + p(\rho) \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \\ -\rho g \end{pmatrix}. \tag{4}$$

To solve this two-phase flow problem numerically (irrespective of the weakly compressible flow condition), we may use a generalization of the classical shock-capturing method designed originally for single phase flows. In this case, it is known in the literature that the principal difficulty in the usual modification is the occurrence of spurious pressure oscillations when two or more fluid components are present in a grid cell (cf. [1,15]). To overcome this difficulty, in this section, we take an approach advocated by one of the the authors [39,40] in that thermodynamically the mixture of the fluids characterized by (2) and (3) is assumed to behave like a typical non-barotropic fluid, and can be modeled by the stiffened gas equation of state,

$$p(\rho, e) = (\gamma - 1) \rho e - \gamma \mathcal{B}. \tag{5}$$

For convenience, we introduce a volume-fraction function $\alpha \in [0,1]$ to represent the type of fluid component within a grid cell. For example, when grid cells contain only the liquid phase we may set $\alpha=1$, and so when grid cells contain only the gas phase we set $\alpha=0$. In case there are some cells cut by the interfaces where $\alpha \in (0,1)$, we then have both of these components occupied by the volume fractions α and $1-\alpha$ for each separately. With this definition of α , the pressure of our two-phase flow problem in all the fluid-component scenarios within a grid cell can be determined straightforwardly by

$$p = \begin{cases} (p_0 + \mathcal{B}) \left(\frac{\rho}{\rho_0}\right)^{\gamma} - \mathcal{B} & \text{if } \alpha = 1\\ (\gamma - 1) \rho e - \gamma \mathcal{B} & \text{if } \alpha \neq 1, \end{cases}$$
 (6)

provided that all the variables appeared there are defined and known a priori.

With the pressure law (6), the governing equations for the proposed two-phase flow problems may be written of the form

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x} (\rho u) + \frac{\partial}{\partial y} (\rho v) = 0,$$

$$\frac{\partial}{\partial t} (\rho u) + \frac{\partial}{\partial x} (\rho u^2 + p) + \frac{\partial}{\partial y} (\rho u v) = 0,$$

$$\frac{\partial}{\partial t} (\rho v) + \frac{\partial}{\partial x} (\rho u v) + \frac{\partial}{\partial y} (\rho v^2 + p) = -\rho g,$$

$$\frac{\partial}{\partial t} (\rho E) + \frac{\partial}{\partial x} (\rho E u + p u) + \frac{\partial}{\partial y} (\rho E v + p v) = -\rho g v \quad \text{if} \quad \alpha \neq 1,$$

$$\frac{\partial \alpha}{\partial t} + u \frac{\partial \alpha}{\partial x} + v \frac{\partial \alpha}{\partial y} = 0,$$
(7)

where the mixtures of γ and \mathcal{B} are computed by

$$\gamma = 1 + \left(1 / \sum_{\iota=1}^{2} \frac{\alpha_{\iota}}{\gamma_{\iota} - 1}\right) \quad \text{and} \quad \mathcal{B} = \left(\sum_{\iota=1}^{2} \alpha_{\iota} \frac{\gamma_{\iota} \mathcal{B}_{\iota}}{\gamma_{\iota} - 1}\right) / \left(1 + \sum_{\iota=1}^{2} \frac{\alpha_{\iota}}{\gamma_{\iota} - 1}\right).$$

Note that in above we have set $\alpha_1 = \alpha$ and $\alpha_2 = 1 - \alpha$ for the volume fractions occupied by the fluid-component 1 and 2, in a respective manner. With a system formulated in this way, there is no problem to compute all the state variables of interest, including the pressure from the equation of state. The initialization of the state variables in (7) for fluid-mixture cells can be made in a standard way as described in [37,38] for numerical simulations.

We note that when the thermodynamical description of the materials of interest is limited by the stability requirement, the speed of sound of the fluid would belong to a set of real numbers, and so hyperbolicity of this two-phase flow model can be verified by standard characteristic analysis. For the ease of the latter reference, it is useful to write (7) into a more compact expression by

$$\frac{\partial q}{\partial t} + \frac{\partial f_1(q)}{\partial x} + \frac{\partial f_2(q)}{\partial y} + B_1(q)\frac{\partial q}{\partial x} + B_2(q)\frac{\partial q}{\partial y} = \psi(q)$$
(8)

with

$$q = [\rho, \rho u, \rho v, \rho E, \alpha]^{T},$$

$$f_{1} = [\rho u, \rho u^{2} + p, \rho u v, \rho E u + p u, 0]^{T},$$

$$f_{2} = [\rho v, \rho u v, \rho v^{2} + p, \rho E v + p v, 0]^{T},$$

$$B_{1} = \text{diag}[0, 0, 0, 0, u],$$

$$B_{2} = \text{diag}[0, 0, 0, 0, v],$$

$$\psi = [0, 0, -\rho g, -\rho g v, 0]^{T}.$$

If we now take the standard time-splitting version of a compressible flow solver for numerical approximation of this two-phase flow model (see [19, 21] for an unsplit quasi-steady version of the approach to deal with the gravitational source terms considered here), in each time step, we would alternate between solving the homogeneous part of the equations with no source terms

$$\frac{\partial q}{\partial t} + \frac{\partial f_1(q)}{\partial x} + \frac{\partial f_2(q)}{\partial y} + B_1(q)\frac{\partial q}{\partial x} + B_2(q)\frac{\partial q}{\partial y} = 0, \tag{9}$$

and solving the ordinary differential equations

$$\frac{\partial q}{\partial t} = \psi(q). \tag{10}$$

In the numerical results shown in Section 5, for comparison purposes a couple of the state-of-the-art explicit methods are used to find approximate solutions for (9), while the second order Runge-Kutta method (cf. [16]) is used to find approximate solutions for (10).

To end this section, it is interesting to note that rather than using the Tait equation of state (3) as the constitutive law for the liquid, we may use the stiffened gas equation of state (5) for that instead. If this is the case, we may simply remove $\alpha \neq 1$ constraint from the energy equation in (7), and take the whole set of equations as a model system for the corresponding two-phase flow problems, see [37] for the details.

3. A PRECONDITIONED COMPRESSIBLE TWO-PHASE SOLVER WITH SURFACE TENSION

Our second physical two-phase flow model assumes again the pressure and velocity equilibrium across interfaces, and is obtained from an asymptotic limit of the non-equilibrium Bear-Nunziato system [2, 14, 27]. If we include the capillarity effect, but ignore the viscous and heat conducting effects as before, the two-dimensional case of the equations can be written analogously as (8) to be of form

$$\frac{\partial q}{\partial t} + \frac{\partial f_1(q)}{\partial x} + \frac{\partial f_2(q)}{\partial y} + B_1(q)\frac{\partial w}{\partial x} + B_2(q)\frac{\partial w}{\partial y} = \psi(q)$$
(11)

with

Here ρ_{ι} denotes the partial density of the fluid component ι , β is a function that takes into account of the compaction effect of the flow (cf. [2, 14, 27]), σ is the surface tension coefficient, κ is the curvature at an interface, and ϕ is the level set function where $\phi = 0$ gives the approximate location of the interface. It is easy

to see that the continuum surface force approach (cf. [4]) has been used in the equations to model the capillarity effect of the problem.

To complete the model, we assume that the equation of state associated to each component ι is governed by the a linearized Mie-Grüneisen equation of state [26] as

$$p(\rho_t, e_t) = (\gamma_t - 1) \left(\rho_t e_t - \rho_t e_t^{\infty} \right) - \gamma_t p_t^{\infty}, \tag{12}$$

where $e_{\iota} = e_{\iota}(\rho_{\iota}, p)$ is the partial internal energy of fluid component ι . Recall that the mixture density, internal energy, and total energy are defined as

$$\rho = \sum_{\iota=1}^{2} \alpha_{\iota} \rho_{\iota}, \quad \rho e = \sum_{\iota=1}^{2} \alpha_{\iota} \rho_{\iota} e_{\iota}, \quad \text{and} \quad \rho E = \rho e + \frac{1}{2} \rho \left(u^{2} + v^{2} \right),$$

respectively. In this case, as before, to obtain the pressure equilibrium for multiphase grid cells, the mixtures of γ , p^{∞} , and e^{∞} can be obtained from the relations:

$$\frac{1}{\gamma - 1} = \sum_{t=1}^{2} \frac{\alpha_t}{\gamma_t - 1}, \qquad \frac{\gamma p^{\infty}}{\gamma - 1} = \sum_{t=1}^{2} \alpha_t \frac{\gamma_t p_t^{\infty}}{\gamma_t - 1}, \qquad \text{and} \qquad \rho e^{\infty} = \sum_{t=1}^{2} \alpha_t \rho_t e_t^{\infty}. \tag{13}$$

It is known in the literature that the major numerical difficulties for the resolution of this system (11) is related to the non-conservative product term for the compaction of the volume fraction. The mathematical analysis shows that, under the constitutive laws (12) and (13), this model is hyperbolic with real eigenvalues and a complete set of the corresponding eigenvectors. Note that the equilibrium sound speed would depend on the asymptotic regime of concern (cf. [14,25,27]), and due to the surface tension effect, the pressure would not be continuous across a material interface, but is having a jump that obeys the Laplace law locally [18].

One of the crucial requirement of a multiphase algorithm is its ability to accurately resolve a wide range of multiphase conditions, from fully compressible to the low Mach regimes. As mentioned in the introduction, the low Mach number setting is a singular asymptotic in compressible flows in which the basic feature of the flow is characterized by a large discrepancy between the magnitude of the acoustic wave-speed and the flow velocity, and the usual time-marching solvers for compressible flow is not adequate to compute the weakly compressible or nearly incompressible flows.

In this section, we are interested in a preconditioned-based method for low Mach flow and the preconditioning is obtained by the introduction of an acoustic fluctuation characteristic time τ associated to "incompressible" pressure that is smaller than the mean flow characteristic time t. Time-marching solutions are then designed from pseudo acoustic-waves obtained by altering time-derivatives of the pressure. This is similar to the procedure used in the pseudo-time (or dual-time) [6,42,45,47] and pseudo-compressibility approaches [31]. The detailed description of our approach can be found in [5].

4. An incompressible two-phase solver

Our third and final physical model for a two-phase flow problem concerns an incompressible flow, and is based on an artificial compressibility approach [8]. In two dimensions, the dimensionless form of the equations

may be written as

$$\frac{\partial p}{\partial t} + \frac{\partial}{\partial x} (\beta u) + \frac{\partial}{\partial y} (\beta v) = 0,$$

$$\frac{\partial u}{\partial t} + \frac{\partial}{\partial x} (u^2 + p) + \frac{\partial}{\partial y} (uv) = \frac{1}{\text{Re}} \nabla^2 u,$$

$$\frac{\partial v}{\partial t} + \frac{\partial}{\partial x} (uv) + \frac{\partial}{\partial y} (v^2 + p) = \frac{1}{\text{Re}} \nabla^2 v - \frac{1}{\text{Fr}} g,$$

$$\frac{\partial \alpha}{\partial t} + \frac{\partial}{\partial x} (u\alpha) + \frac{\partial}{\partial y} (v\alpha) = 0,$$
(14)

where β is the coefficient of artificial compressibility, Re is the Reynold's number, and Fr is the Froude number. To solve (14) numerically, we use a third-order MUSCL (Monotone Upstream-centered Scheme for Conservation Laws) to discrete the convection terms, a second-order finite volume method to discrete the viscous terms, and the two-step Runge-Kutta time integration is used for time marching. For the unsteady problem, we apply the Crank-Nicolson method to the original incompressible Navier-Stokes equations, and then use pseudo-time and the artificial compressibility method to solve the unsteady problems, see [22] for the details. Similar methods of this kind can be found, for example, in the work done by Wackers and Koren [49] and Qian et al. [22].

5. Numerical results

We now present results for a sample low Mach test problems obtained using various numerical approaches described in the previous sections.

5.1. Broken dam problem

To begin with, we consider the well-known broken dam problem of Martin and Moyce [24]. Here the initial condition we take consists of a water column with 0.06m wide and 0.12m high that is confined to the left in a rectangular container of size $[0,0.5] \times [0,0.15]$ m² with air inside in the standard atmospheric condition, see Fig. 1 for illustration. It is clear that under the effect of the gravity, the water column would collapse subsequently, and the leading water front would move to the right of the domain.

To solve this problem numerically, we first consider a compressible homogeneous flow model described in Section 2, and employ a standard high-resolution wave propagation method for the homogeneous part of the equations (cf. [20, 40]). Numerical results for this problem with three different mesh sizes 100×30 , 200×60 , and 400×120 are presented in Figs. 1 and 2. That is, to demonstrate the basic feature of the temporal behavior of the water column, Fig. 1 shows the pseudo-color plots of the volume fraction at six different times t=0, 0.066, 0.109, 0.164, 0.222,and 0.281s. From the figure, when the meshes are of size 100×30 and 200×60 , the existence of a kink-like structure near the corner of the water column is an anomalous behavior which should be removed by a chosen numerical method. In addition, to validate the computed solutions, Fig. 2 shows the time history of the water column height at the left boundary and also the leading water front position at the bottom boundary, where the numerical results are in comparison with the experimental results provided by Martin and Moyce [24]. Note that even though the Mach number is typically very small over the time period under consideration, the computed solutions are in reasonable agreement with the experimental results as the mesh is refined. To carry out the above computation we have used the Courant number 0.9, and employed solid wall boundary conditions to all the boundaries. The material constants are for the air: $\rho = 1.2 \text{ kg/m}^3$, $\gamma = 1.4$, and for the water: $\rho_0 = 10^3 \text{ kg/m}^3$, $p_0 = 1 \text{ atm}$, $\gamma = 7$, and $\mathcal{B} = 3000 \text{ atm}$. The gravitational constant we take is $g = 9.81 \text{ m/s}^2$.

Before presenting numerical results obtained using the other mathematical models, we pause to examine the effect of the equation of state for the water in this compressible two-phase flow model, and for comparison purposes, we also perform the computations using the AUSM-based third order ENO and WENO methods [30].

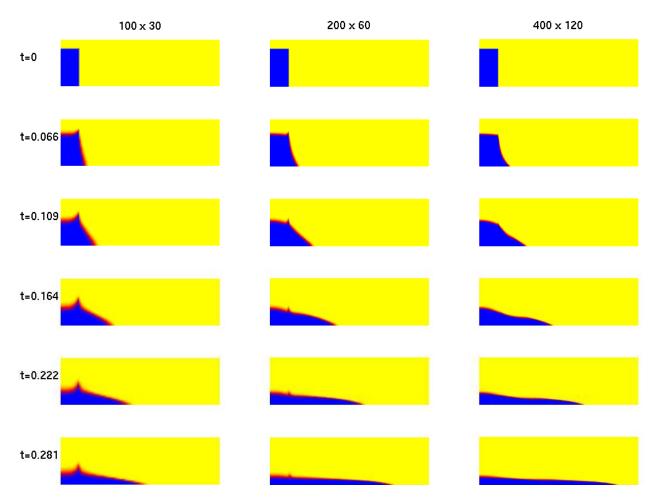


FIGURE 1. Pseudo-color plots of volume fraction for the broken dam problem obtained using a compressible homogeneous model described in Section 2, and a high-resolution wave propagation method. Results are shown at six different times $t=0,\,0.066,\,0.109,\,0.164,\,0.222,$ and 0.281s with three different mesh points $100\times30,\,200\times60,$ and $400\times120.$

Figure 3 gives the validation results for the time history of the water column height at the left boundary and also the leading water front position at the bottom boundary, where the stiffened gas equation of state (5) is used for the water with $\gamma = 4.4$, $\mathcal{B} = 6000$ atm, and the grid size is 400×120 . From the figure, it is easy to observe that the equation-of-state effect is not significant for this problem, and qualitatively there is not much difference between the results obtained using the various numerical methods consider here.

Our second attempt to solve this problem numerically is to employ a preconditioned compressible flow solver described in Section 3 which is an implicit method with Courant number 20 and Gauss-Seidel solver (50 iterations maximum) included. As before, in Fig. 4, we validate the solutions by comparing them to the experimental results on the water column height at the left boundary and also the leading water front position at the bottom boundary. Reasonable agreement between the results are observed also as the mesh is refined. It should be mentioned that from quite a few numerical evidences (not shown here) the surface-tension effect at the air-water interface does not play an important role at the early stage of the motion of this collapse of water column. It would be a significant effect however right after the eventual breaking of the water column at a later time due to

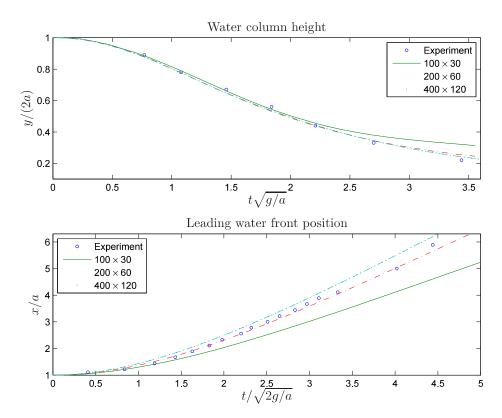


FIGURE 2. Comparison between numerical solutions shown in Fig. 1 and experimental results presented in Martin and Moyce [24]. On the top, results on the water column height are shown, and on the bottom, results on the leading water front position are shown. A mesh-refinement sequence $2^{i}(100 \times 30)$ for i=1,2,3 are used for comparison. Here we have a=0.06m and g=9.81 m/s².

the wave collision to the container's right and top boundaries. Note that the material constants we use here are for the air: $\rho=1$ kg/m³, $\gamma=1.4$, $p^{\infty}=0$, $e^{\infty}=0$, and for the water: $\rho=10^3$ kg/m³, $\gamma=4.4$, $p^{\infty}=6000$ atm, and $e^{\infty}=0$.

Our third and final attempt to solve this problem numerically is to employ an incompressible flow solver based on the artificial compressibility approach described briefly in Section 4. In the method, a dual-time strategy is used with a sub-iteration step included to achieve the divergence-free flow condition at every physical time step. In Fig. 5, we present the validation results on the water column height at the left boundary and also the leading water front position at the bottom boundary, with a 200×60 mesh points. It is interesting to see that this incompressible flow solver works rather good for this broken dam problem. Note that results obtained using compressible flow solvers as shown in Figs. 2 and 4 are included in the graph also as for comparison. Here the mesh point is 200×60 . Here the parameters we use in the run for the method are $\beta = 1$, Re = 200, Fr = 1, and the density ratio between the water and the air is 1000.

Table 1 gives the computational cost in CPU time for all the runs presented in this subsection. It is clear that the preconditioned compressible flow solver described in Section 3 gives a somewhat better performance than the other two numerical approaches. It is also interesting to see that the standard compressible flow solver (with wave propagation method) works faster than the AUSM-based method and the incompressible flow solver, when viewing on the same level of the grid size.

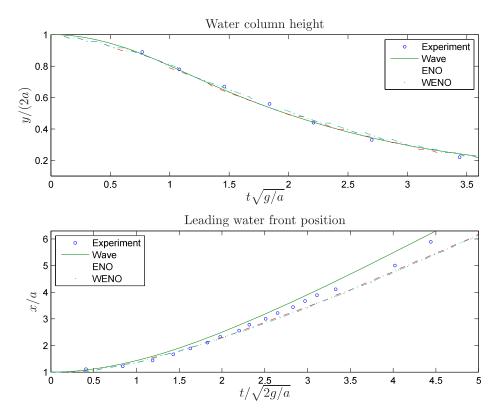


FIGURE 3. Numerical results for the study of the equation-of-state effect to the broken dam problem with a 400×120 grid. The stiffened gas equation of state (5) is used in the compressible homogeneous model described in Section 2 when we carry out the computation using the AUSM-based third order ENO and WENO methods, while as in Fig. 2 the Tait equation of state (3) is used with the wave propagation method.

Table 1. Computational cost for a broken dam problem measured in CPU time

| Method | Mesh | CPU time (sec) | CPU type |
|---------------------------------|------------------|----------------|-------------------------|
| Compressible solver(wave-based) | 100×30 | 492 | AMD Dural-Core Opteron |
| | 200×60 | 3782 | $2220 2.8 \mathrm{GHz}$ |
| | 400×120 | 31783 | |
| Compressible solver(AUSM-ENO) | 400×120 | 105200 | IBM P595 |
| Compressible solver(AUSM-WENO) | 400×120 | 170000 | |
| Precond. compressible solver | 100×30 | 352 | Intel Core 2 Duo 3.0GHz |
| | 200×60 | 2453 | |
| | 400×120 | 21780 | |
| Incompressible solver | 200×60 | 9804 | Intel Pentium4 3.4GHz |

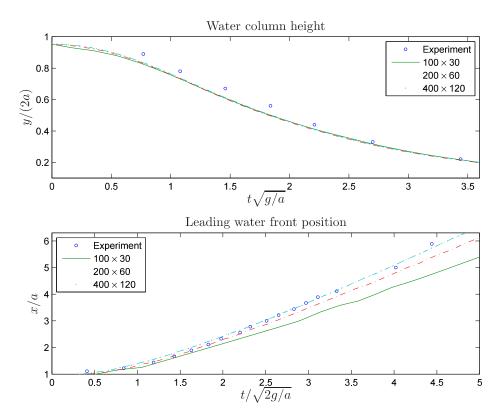


FIGURE 4. Comparison between numerical solutions obtained using a preconditioned compressible flow solver described in Section 3 and experimental results for the broken dam problem. The graph of the solutions are displayed in the same manner as in Fig. 2.

5.2. Wave breaking problem

We are next concerned with a wave breaking problem that has been studied extensively in the literature, see Helluy et al. [13] and the references cited therein. The initial condition is a rightward-moving solitary wave which can be computed by the Tanaka method [43] traveling toward a step-like reef on the right, see Fig. 6 for the schematic setup, and see [13] for more details.

As a first attempt to solve this problem numerically, we use the same explicit compressible flow solver with the high-resolution wave propagation method as before for the broken dam problem, and we present the results in Figs. 7 and 8 where the programs are run with three different mesh sizes 400×50 , 800×100 , and 1600×200 . From the pseudo-color plots of the volume fraction shown in Fig. 7 at four different times t=1.2, 1.35, 1.6, and 1.8s, the breaking of the water wave is clearly seen as the mesh is refined. To validate the computed solutions, Fig. 8 shows the time history of the elevation of the water surface at the three different gauges P2, P3, and P4 which are located at x=2m, 2.515m, and 3.02m, respectively. Note that for this problem, experimental time evolution of the water elevations are available at http://helluy.univ-tln.fr/soliton.htm. From the graph, it is easy to notice the sensible agreement of the results as the mesh turns finer.

We next continue solving this wave breaking problem by the same preconditioned compressible flow solver as done before. Figure 9 gives the Schlieren-type images of volume fraction at four different time $t=1.2,\,1.35,\,1.6,\,$ and 1.8s where a 1500×200 mesh points is used in the computation. As far as the qualitative global wave structure is concerned, we notice reasonable resolution of the solution as in comparison with the results shown in Fig. 7 (the 1600×200 grid case) and also in [13]. To validate the results, in Fig. 10, we again present the

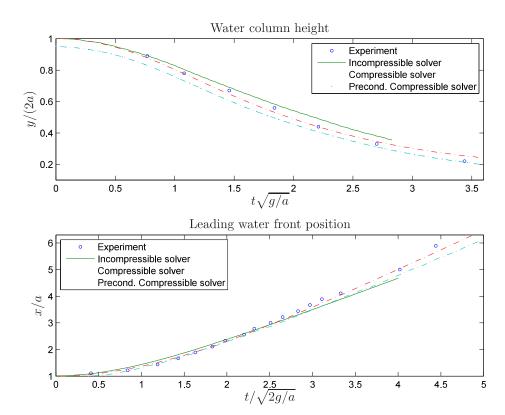


FIGURE 5. Comparison between numerical solutions obtained using an incompressible flow solver described in Section 4 and experimental results for the broken dam problem. Results obtained using compressible flow solvers as shown in Figs. 2 and 4 are included in the graph also. Here the mesh point is 200×60 .

time history of the elevation of the water surface at the three different gauges P2, P3, and P4, noticing good results with the experimental data and also the results obtained using the explicit compressible solver.

Table 2 gives the computational cost in CPU time for all the runs presented in this subsection. It is clear that the implicit preconditioned compressible flow solver now out-performs the explicit wave-propagation based compressible solver.

Table 2. Computational cost for a wave breaking problem measured in CPU time

| Method | Mesh | CPU time (sec) | CPU type |
|---------------------------------|-------------------|----------------|-----------------------------|
| Compressible solver(wave-based) | 400×50 | 6756 | AMD Dural-Core Opteron |
| | 800×100 | 55253 | $2220 2.8 \mathrm{GHz}$ |
| | 1600×200 | 476429 | |
| | | | |
| Precond. compressible solver | 1500×200 | 146400 | Intel Quad Core Xeon 3.0GHz |
| | | | |

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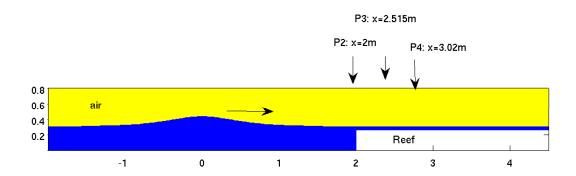


FIGURE 6. Initial setup for the wave breaking problem.

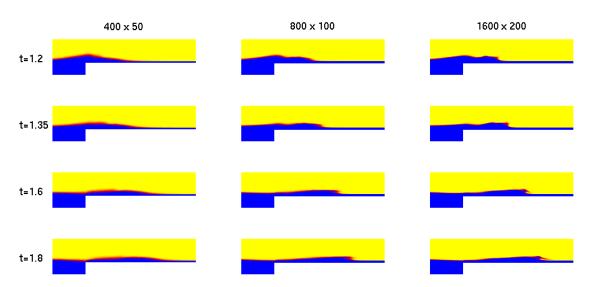


FIGURE 7. Pseudo-color plots of volume fraction for the wave breaking problem obtained using the same explicit compressible flow solver as shown in Fig. 1 for the broken dam problem. Results are shown at four different times $t=1.2,\,1.35,\,1.6,\,$ and 1.8s with three different mesh points $400\times50,\,800\times100,\,$ and $1600\times200.$

6. Conclusion

To assess the feasibility of the existing approaches for numerical resolution of low Mach two-phase flow problems, in the work, we have taken three popular solution techniques into consideration, that is, a standard

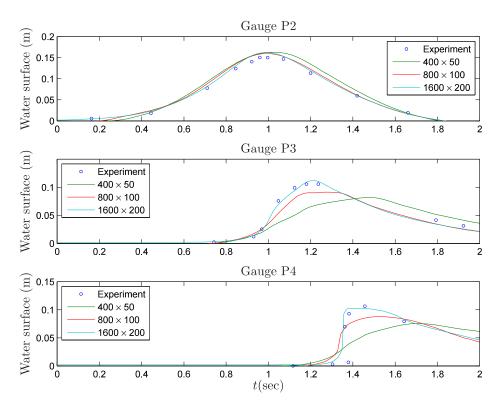


FIGURE 8. Comparison between the computed water surface for the solutions shown in Fig. 7 at three different gauges with experimental results presented in Yasuda *et al.* [52]. A mesh-refinement sequence $2^{i}(400 \times 50)$ for i = 1, 2, 3 are used for comparison.

explicit compressible solver, an implicit preconditioned compressible solver, and an implicit incompressible solver based on artificial compressibility. Numerical results presented here for the two benchmark tests; the broken dam and wave breaking problems, indicate good potential to achieve highly accurate results by many of the aforementioned methods. If efficiency of the method is taken into account, the implicit preconditioned compressible solver is a better choice of the method which is as expected. Future work is to examine more implicit strategies with preconditioned or not preconditioned in the hope to devise a robust method for a wide variety of low Mach multiphase flow problems of practical importance.

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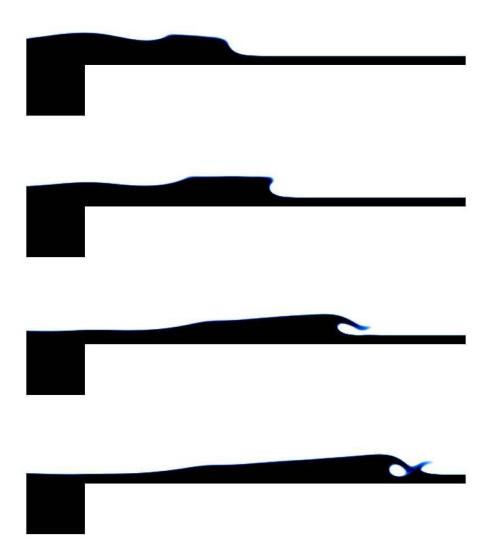


FIGURE 9. Schlieren-type images of volume fraction for the wave breaking problem obtained using the same implicit preconditioned compressible flow solver as shown in Fig. 4 for the broken dam problem. Results from the top to bottom are shown at four different times t=1.2, 1.35, 1.6, and 1.8s with mesh points 1500×200 .

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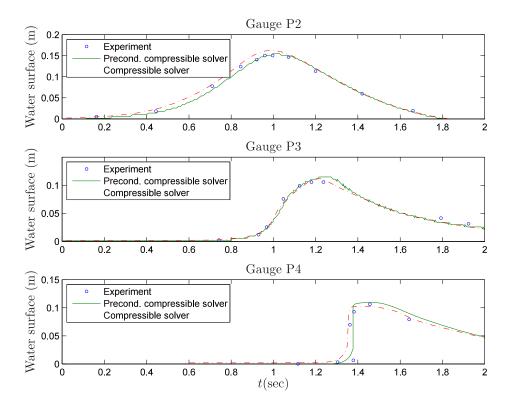


FIGURE 10. Comparison between the computed water surface for the solutions shown in Fig. 9 at three different gauges with experimental results presented in Yasuda *et al.* [52]. Results obtained using compressible flow solvers as shown in Figs. 8 is included in the graph also. Here the mesh point is 1500×200 .

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